



Ventilated window with integrated PCM solar collector

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PCM ventilation system

- ▶Background
- ▶System descriptions
- ▶Case studies
- ▶Discussions



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Building energy consumption states



of total EU energy consumption used to heat and cool our buildings



of the EU's carbon emissions comes from buildings



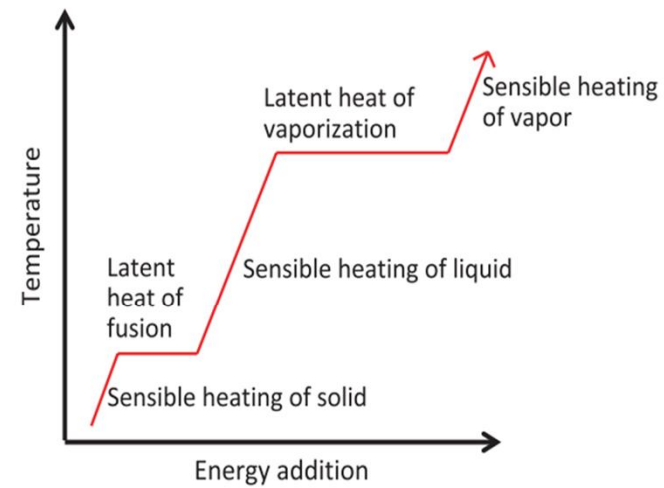
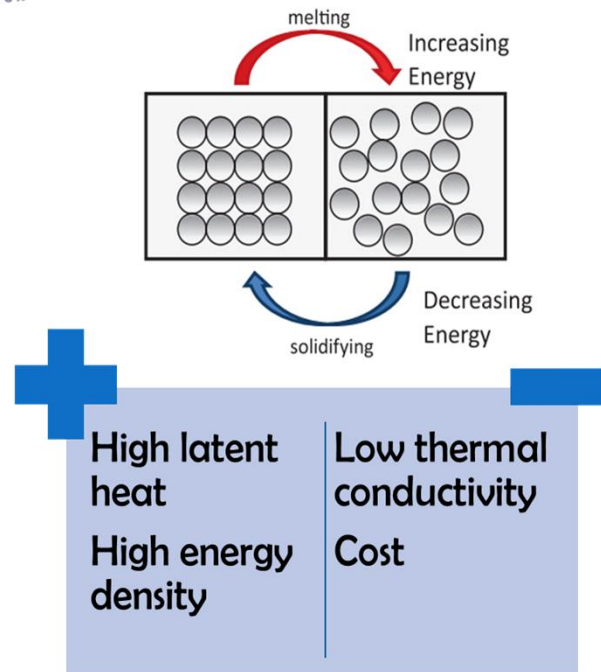
of an average household's energy bill spent on heating and cooling



spent by EU citizens on space and water heating per year

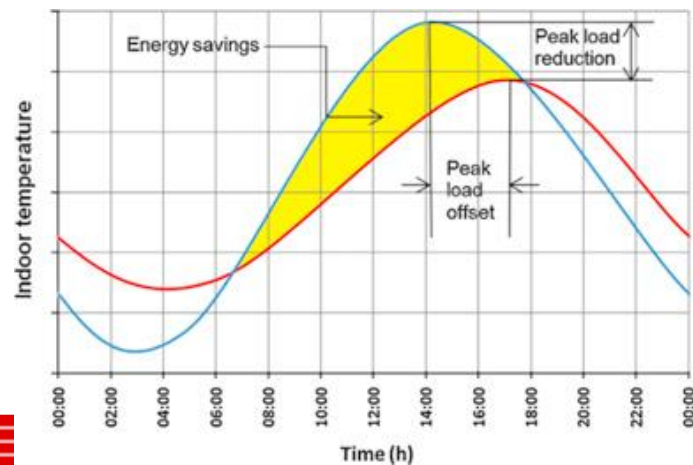
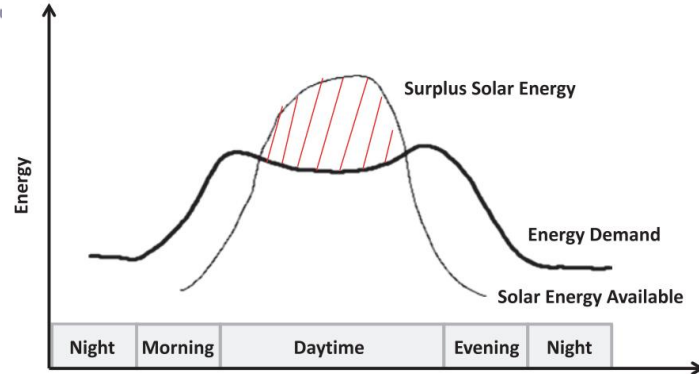
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PCM introduction



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Thermal Energy Storages in buildings

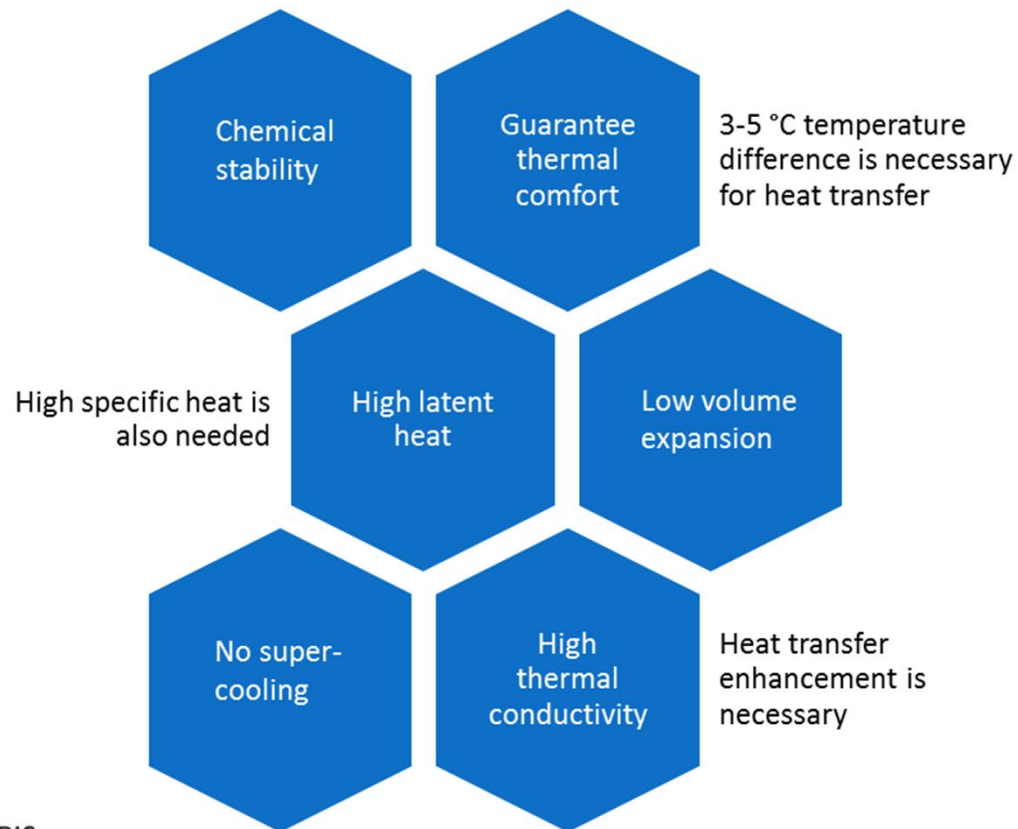


Thermal energy storage(TES):

- **Peak hour shifting** - saving money
- **Heat storage/night cooling** - decrease facility size and cost
- **Isothermally phase change process** – good for comfort

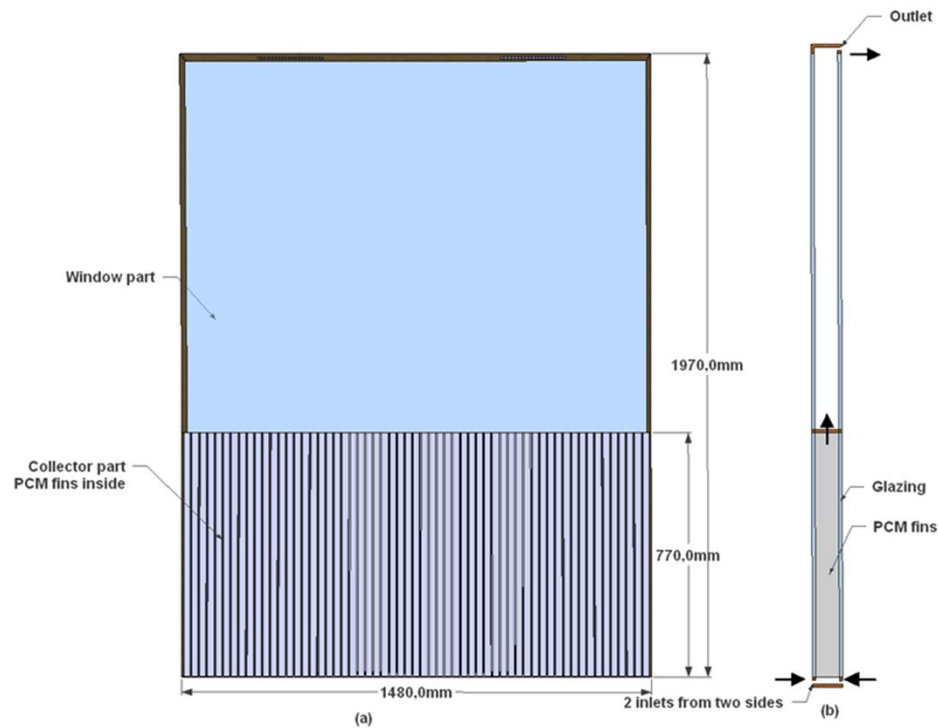
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Select of PCMs in Building application



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PCM heat exchanger +ventilated Window



-What's the importance of PCM in thermal mass and light weight buildings?

-What's the ventilation system with PCM heat exchanger?

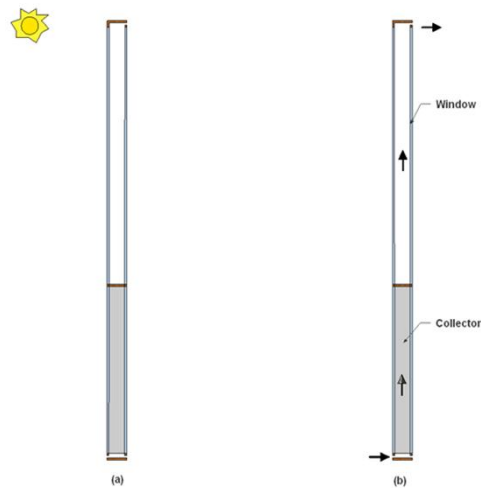
-What are the functions of this PCM ventilated system?

-What's the advantages of this system?

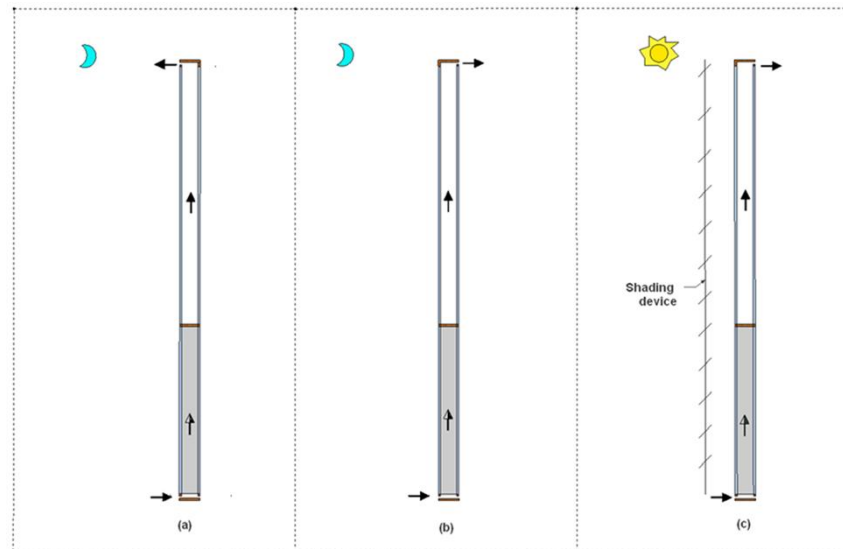
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Control Strategies

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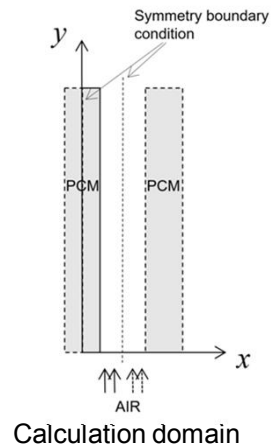


Winter operation modes



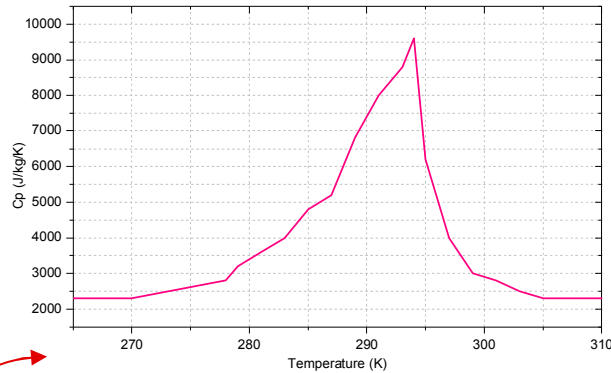
Summer operation modes

Methods



Energy equation:

$$\frac{\partial T}{\partial t} = \frac{\lambda}{\rho C_p(T)} \nabla^2 T$$



Fictive heat capacity

Boundary conditions:

$$-n \cdot q = 0 \quad \text{for symmetry heat transfer boundary condition}$$

$$u \cdot n = 0 \quad \text{for symmetry fluid property boundary condition}$$

$$T(x, y, 0) = 300.15\text{K}$$

$$T(x, 0, t) = 273.15\text{K}$$

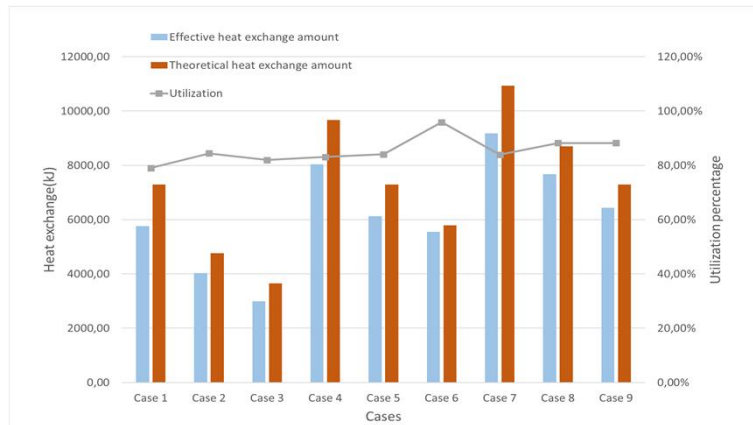
$$Q(x, y, t) = 106 \text{ m}^3/\text{h}$$

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Case study 1

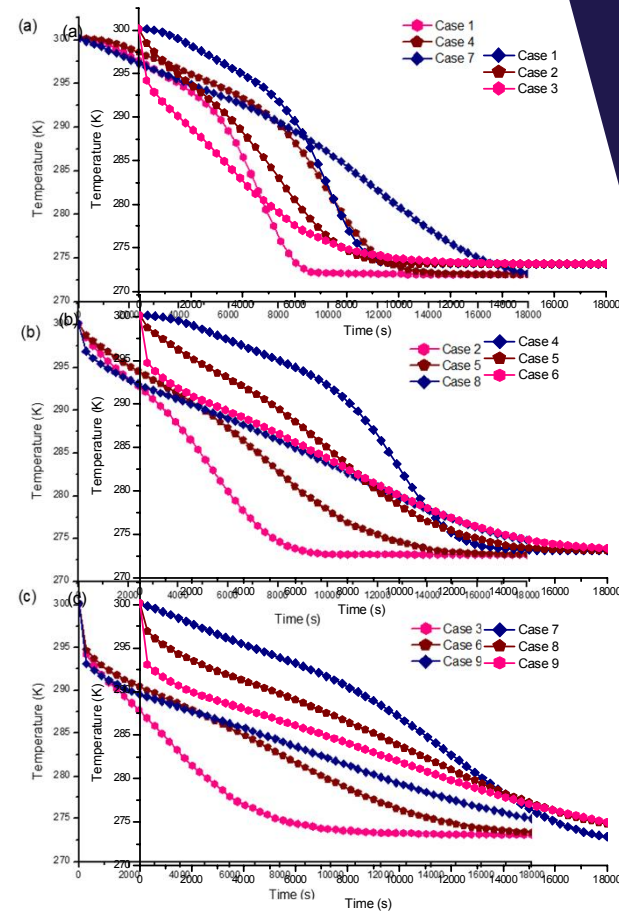
Case study with different fin thickness and air gap thickness

Case	Fin thickness (mm)	Air gap thickness (mm)	Fin number	Fin depth (mm)	Air flow rate (m ³ /h)	Air velocity in gap (m/s)	Total fin surface area (m ²)	Total PCM volume (m ³)
1	5	5	106			0.75	22.30	0.052
2	5	10	70			0.56	14.73	0.034
3	5	15	53			0.50	11.15	0.026
4	10	5	70			1.11	15.69	0.069
5	10	10	53	75	106	0.75	11.88	0.052
6	10	15	42			0.62	9.42	0.041
7	15	5	53			1.50	12.62	0.078
8	15	10	42			0.93	10.00	0.062
9	15	15	35			0.76	7.86	0.052



Heat exchange amount and PCM utilization of 9 cases

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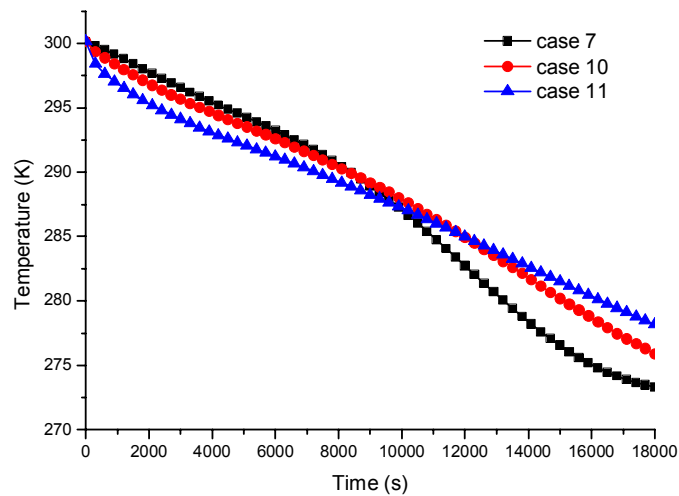




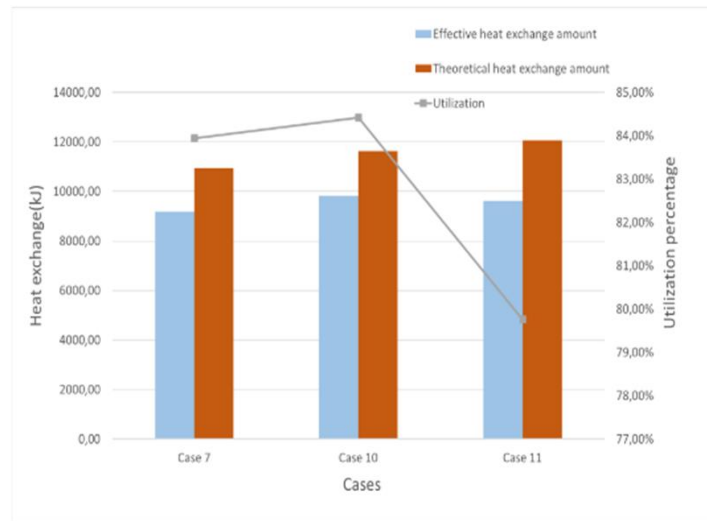
Case study 2

Case study with increased plate thickness

Case	Fin thickness (mm)	Air gap thickness (mm)	Fin depth (mm)	Air flow rate (m ³ /h)	Air velocity in gap (m/s)	Total fin surface area (m ²)	Total PCM volume(m ³)
7	15	5			1.50	12.62	0.078
10	20	5	75	106	1.78	10.58	0.083
11	25	5			2.12	9.30	0.086



Outlet air temperature of 3 cases



Heat exchange amount and PCM utilization of 3 cases

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Case study 3

Orthogonal basis analysis

Experiments	Thickness of PCM plates(D, mm)	Temperature difference between inlet air and phase change temperature(T, °C)	Air flow rate(Q, m/m3)	Discharge time(t, h)
1	10	6	100	8,66
2	10	6	150	7,62
3	10	10	100	7,48
4	10	10	150	6,52
5	15	6	100	11,21
6	15	6	150	9,82
7	15	10	100	9,50
8	15	10	150	8,22

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Source	Type III Sum of Squares	df	Mean Square	F	Sig.
Corrected Model	15,793 ^a	6	2,632	23397,000	,005
Intercept	595,643	1	595,643	5294601,000	,000
D	8,968	1	8,968	79712,111	,002
Q	2,726	1	2,726	24232,111	,004
T	3,906	1	3,906	34720,111	,003
DQ	,056	1	,056	498,778	,028
TQ	,005	1	,005	40,111	,100
DT	,133	1	,133	1178,778	,019
Error	,000	1	,000		
Total	611,436	8			
Corrected Total	15,793	7			

a. R Squared = 1,000 (Adjusted R Squared = 1,000)

Analysis of variance



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Disussions

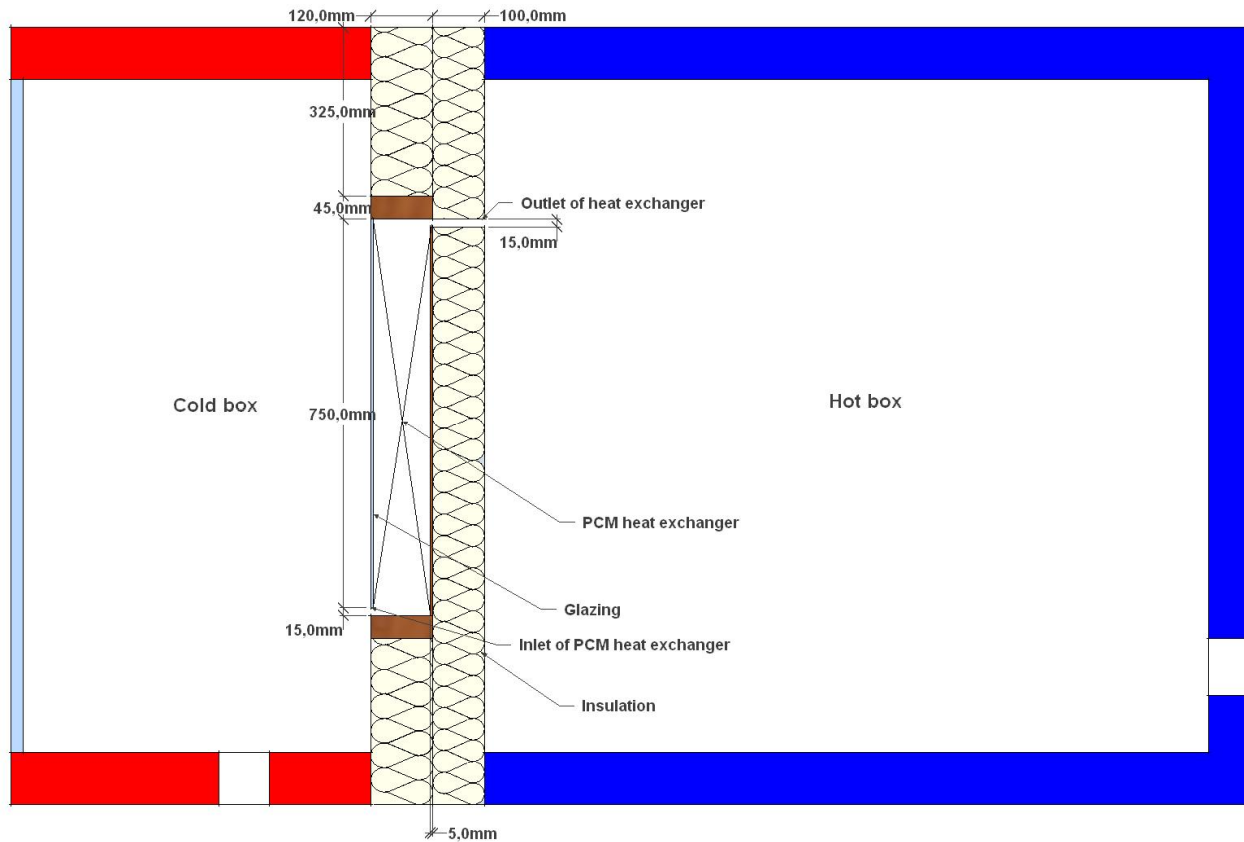
- 2 stages in discharge processes(surface domain stage and volume domain stage).
- Larger plate thickness and smaller air gap thickness result in higher heat exchange amount.
- There is limitation of the plate thickness.
- Mechanical ventilation is needed only in cases with air gap thickness equal to 5mm.
- The system has the potential to completely or partly substitute the air-conditioning and heating system.

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Future works



Experimental verification of the heat transfer process

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Thank you!

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